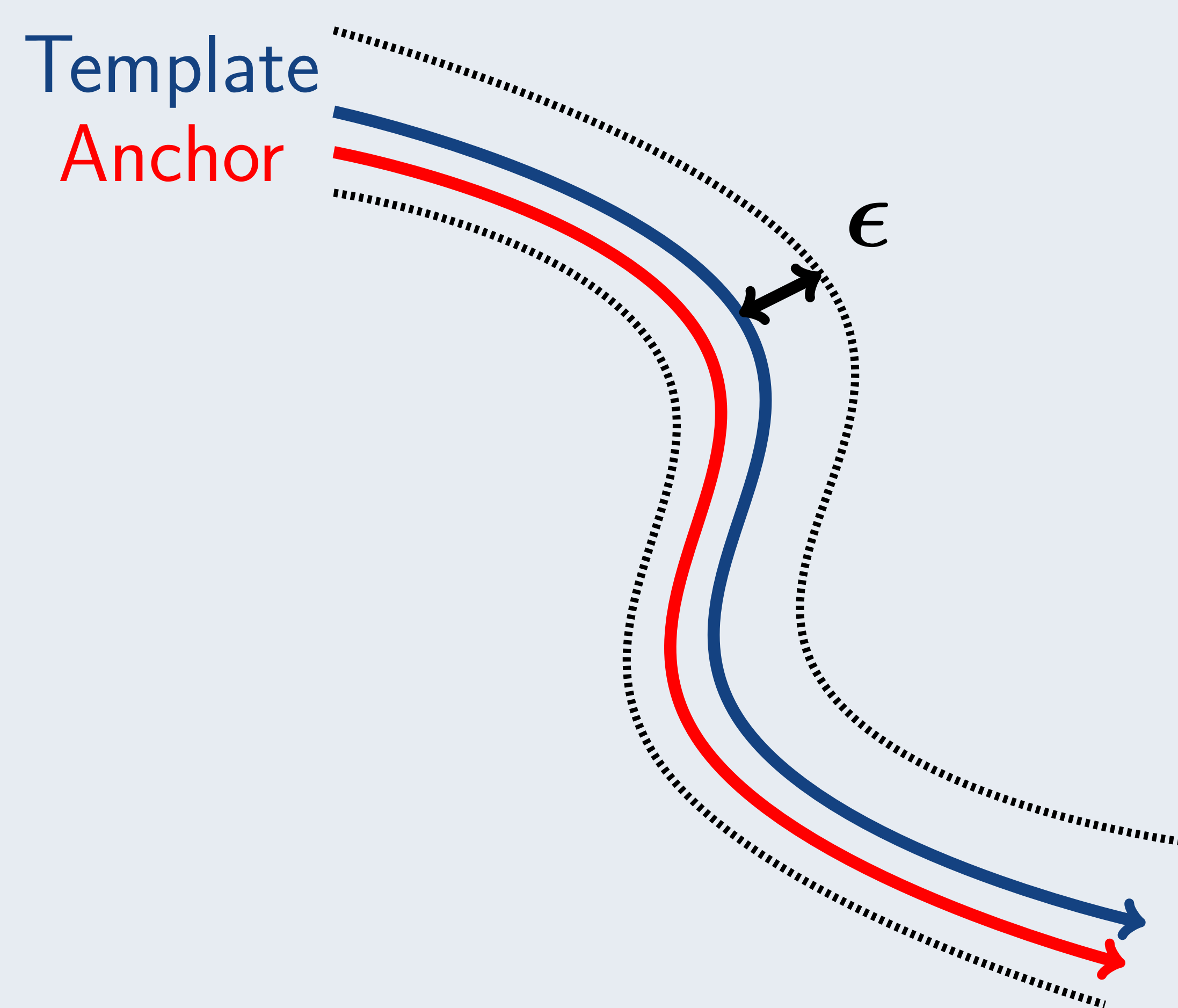


Formal Connections Between Template and Anchor Models

Vince Kurtz, Rafael Rodrigues da Silva, Patrick M. Wensing, and Hai Lin

What is Approximate Simulation?

Ensures that the anchor (whole-body) model can track the template with ϵ precision:



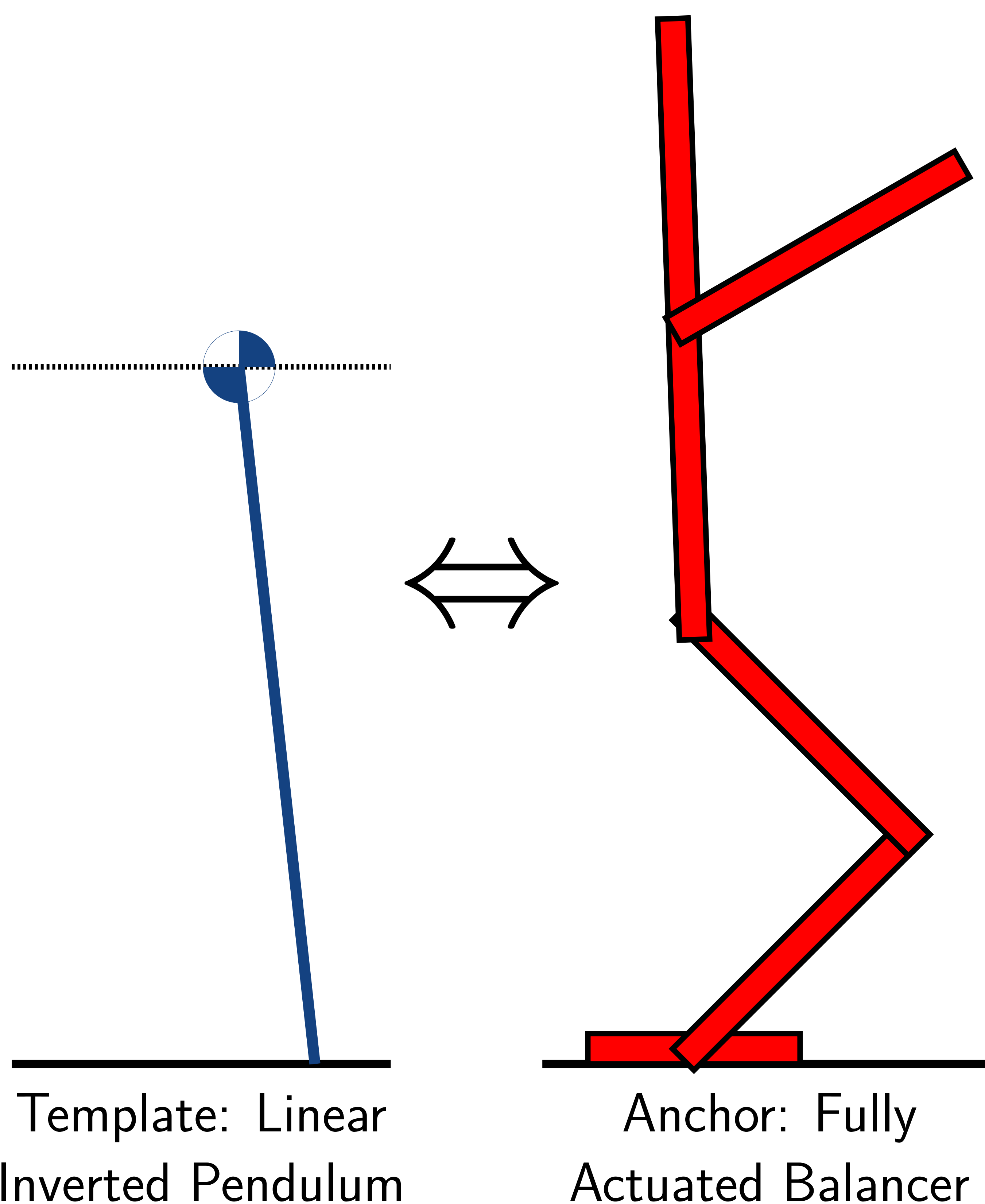
Certified with a **Lyapunov-like simulation function**:

$$\mathcal{V}(\mathbf{x}_1, \mathbf{x}_2) = \sqrt{(\mathbf{x}_1 - \mathbf{P}\mathbf{x}_2)^T \mathbf{M}(\mathbf{x}_1 - \mathbf{P}\mathbf{x}_2)}$$

Interface determines controls that track the template.

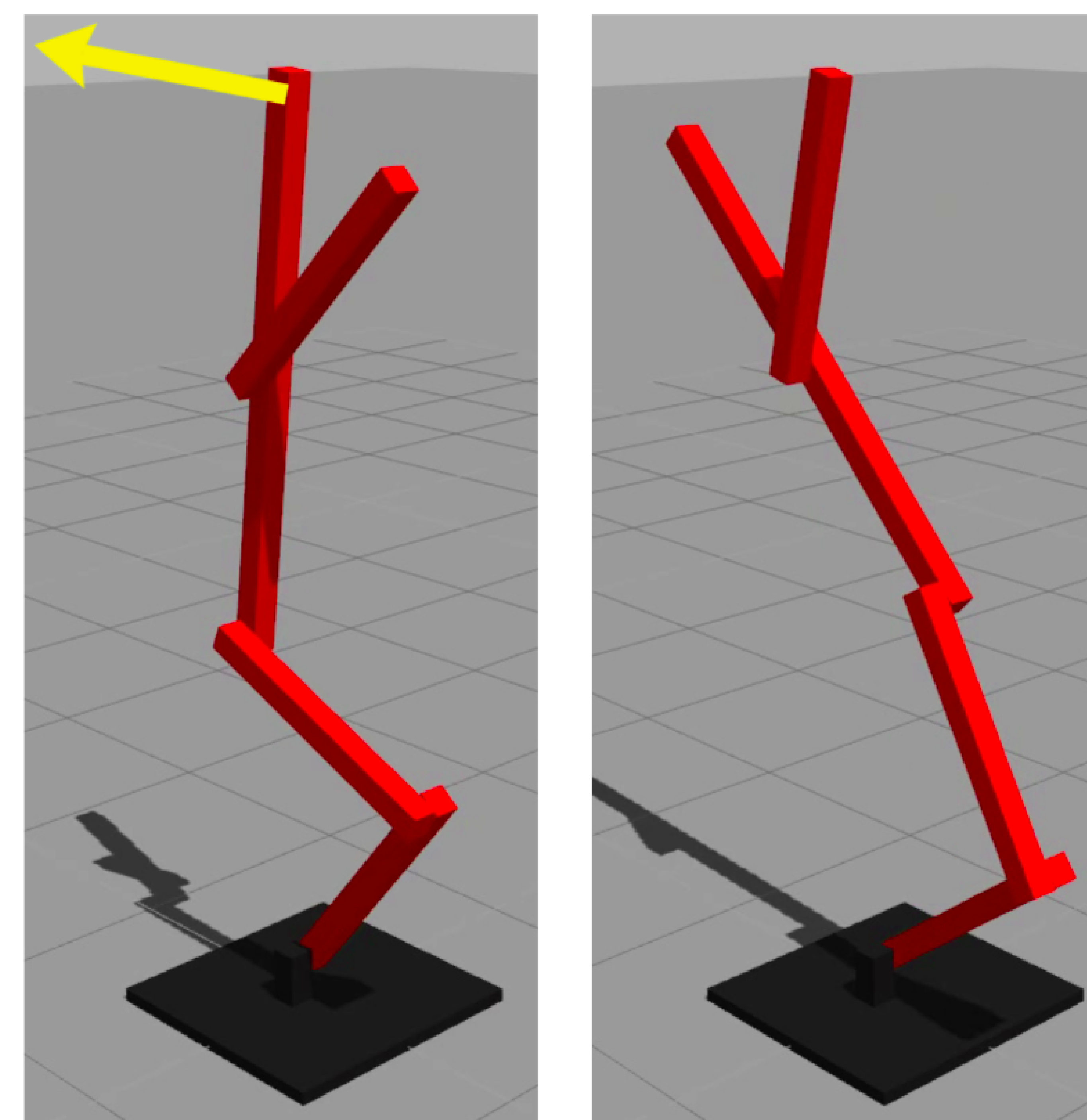
$$\mathbf{u}_1 = \mathbf{R}\mathbf{u}_2 + \mathbf{Q}\mathbf{x}_2 + \mathbf{K}(\mathbf{x}_1 - \mathbf{P}\mathbf{x}_2),$$

We establish an **Approximate Simulation Relation** between the **Linear Inverted Pendulum** and a **Planar Balancer**.



Template: Linear Inverted Pendulum

Anchor: Fully Actuated Balancer



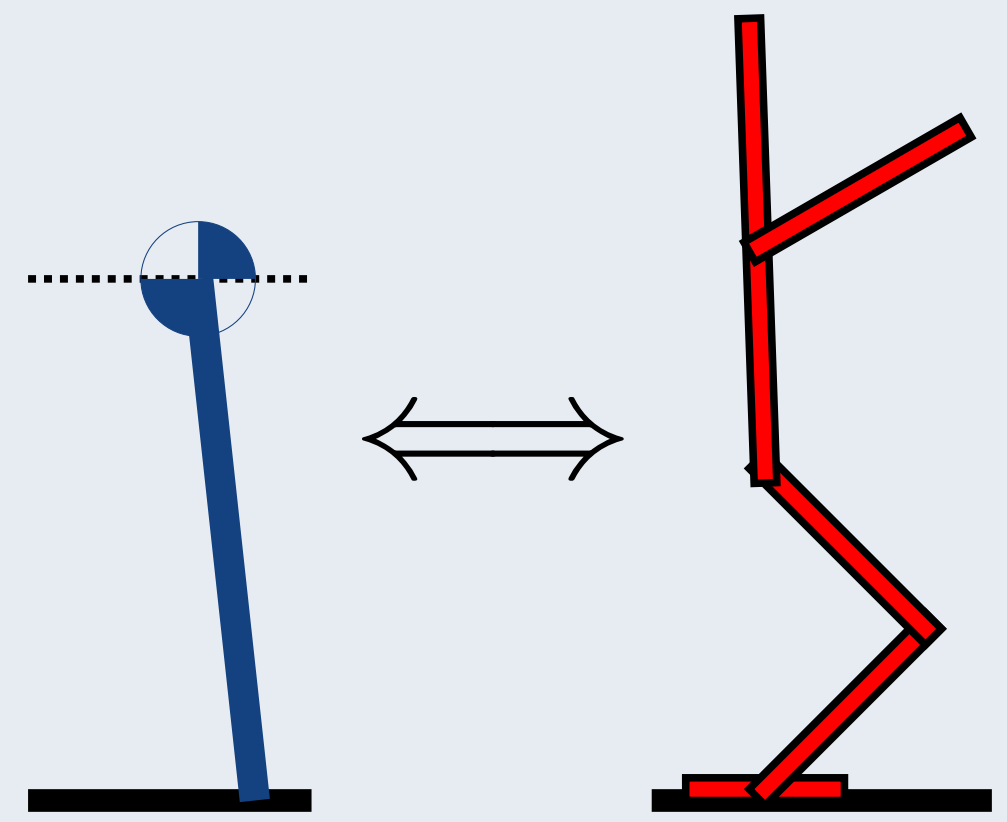
Our approach enables **better push recovery** by allowing centroidal momentum to vary.

Formal Connections Between Template and Anchor Models

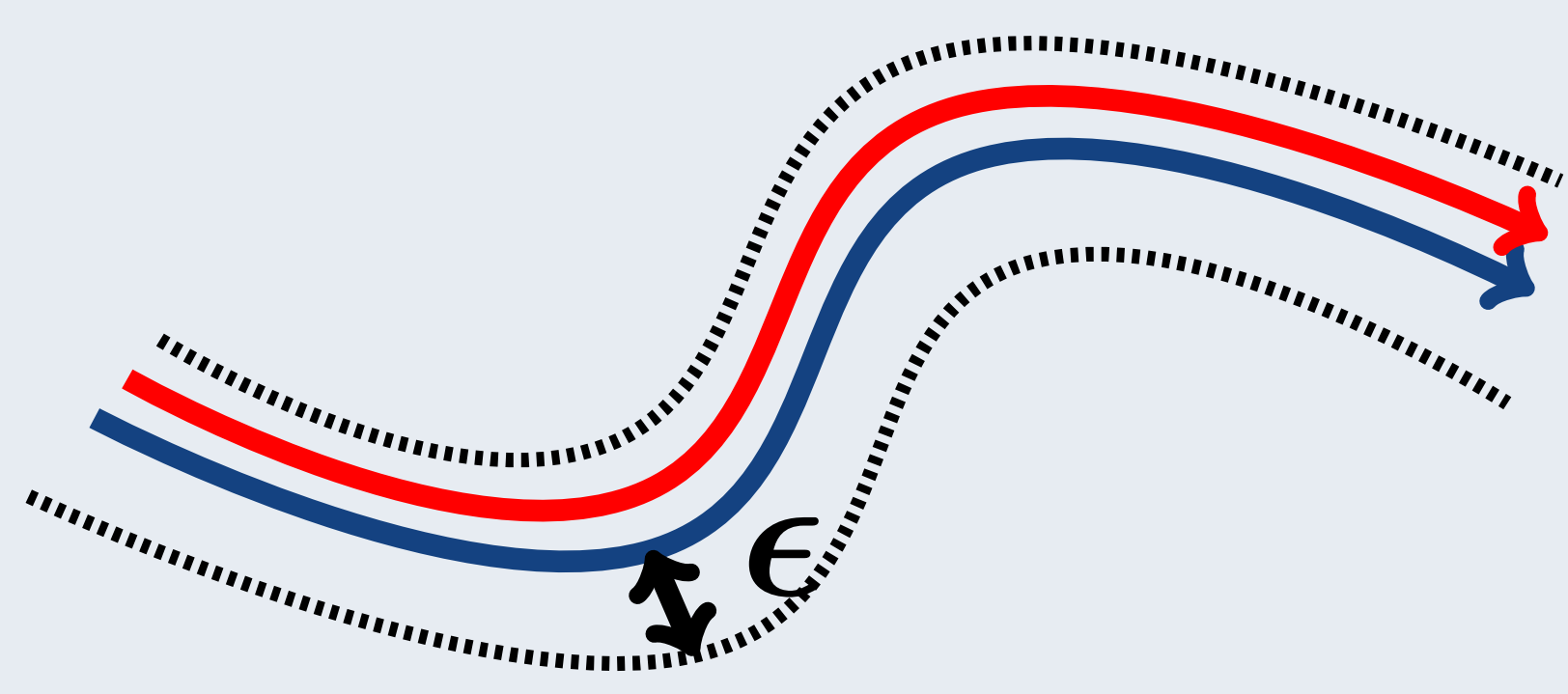
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Summary

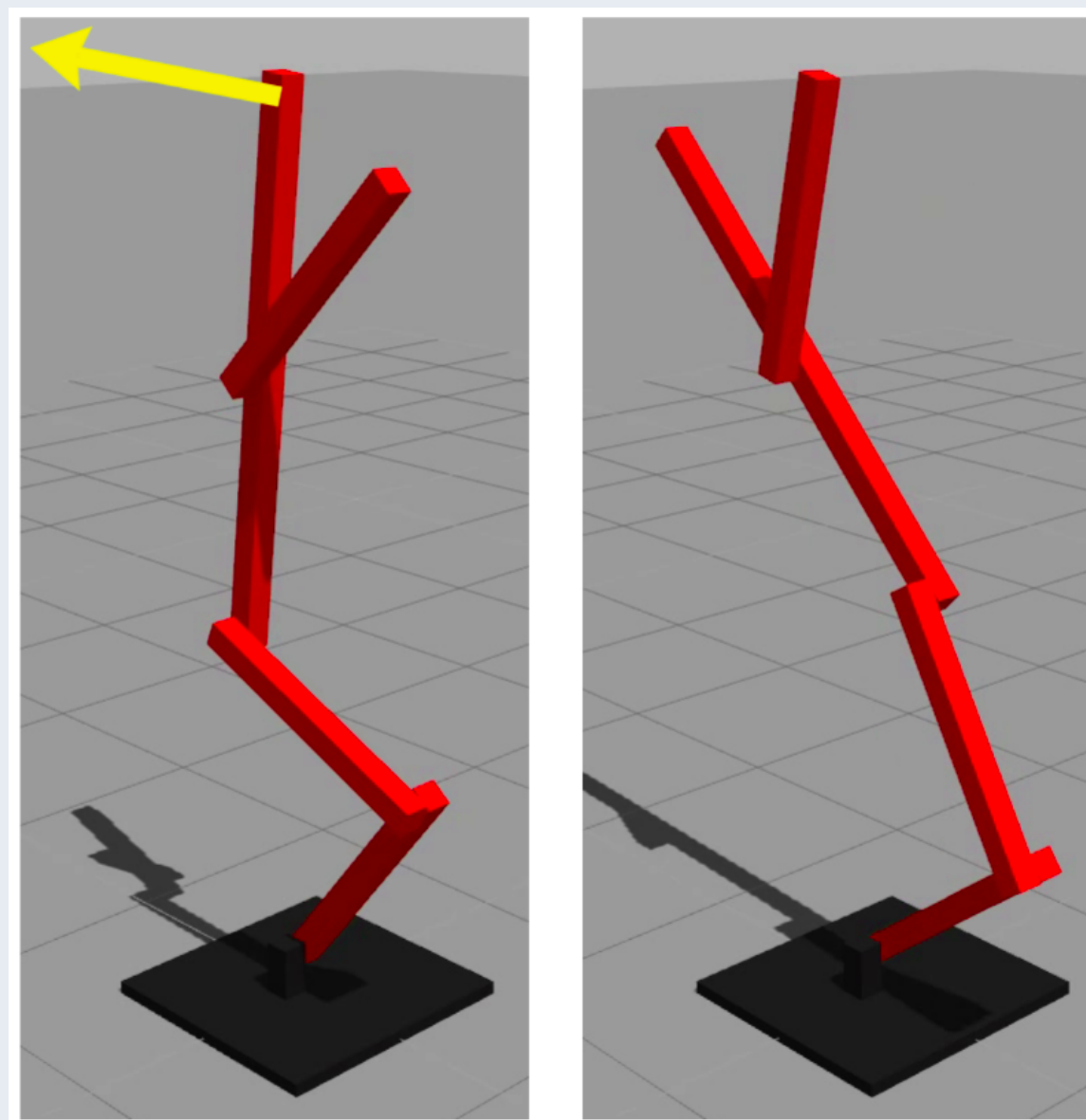
Reduced-order models are useful, but **no formal connections** \Rightarrow **no guarantees**.



We establish an **Approximate Simulation Relation** between the **Linear Inverted Pendulum** and a **Planar Balancer**.



An A.S. relation ensures that the anchor can **track the template with ϵ precision**.



This enables **better push recovery** by allowing centroidal momentum to vary.

Approximate Simulation Relations

Complex anchor system:

$$\Sigma_1 : \begin{cases} \dot{\mathbf{x}}_1 = f_1(\mathbf{x}_1, \mathbf{u}_1) \\ \mathbf{y}_1 = g_2(\mathbf{x}_1) \end{cases}$$

Simpler template system:

$$\Sigma_2 : \begin{cases} \dot{\mathbf{x}}_2 = f_2(\mathbf{x}_2, \mathbf{u}_2) \\ \mathbf{y}_2 = g_2(\mathbf{x}_2) \end{cases}$$

Lyapunov-like Simulation Function:

$$\mathcal{V}(\mathbf{x}_1, \mathbf{x}_2) \geq \|g_1(\mathbf{x}_1) - g_2(\mathbf{x}_2)\|$$

Interface:

$$\mathbf{u}_1 = u_{\mathcal{V}}(\mathbf{u}_2, \mathbf{x}_1, \mathbf{x}_2)$$

For $\gamma(\|\mathbf{u}_2\|) < \mathcal{V}(\mathbf{x}_1, \mathbf{x}_2)$,

$$\frac{\partial \mathcal{V}}{\partial \mathbf{x}_2} f_2(\mathbf{x}_2, \mathbf{u}_2) + \frac{\partial \mathcal{V}}{\partial \mathbf{x}_1} f_1(\mathbf{x}_1, u_{\mathcal{V}}(\mathbf{u}_2, \mathbf{x}_1, \mathbf{x}_2)) < 0.$$

Approximate Simulation for Linear Systems

Difficult to find \mathcal{V} , $u_{\mathcal{V}}$ in general, but for linear systems...

$$\mathcal{V}(\mathbf{x}_1, \mathbf{x}_2) = \sqrt{(\mathbf{x}_1 - \mathbf{P}\mathbf{x}_2)^T \mathbf{M} (\mathbf{x}_1 - \mathbf{P}\mathbf{x}_2)}$$

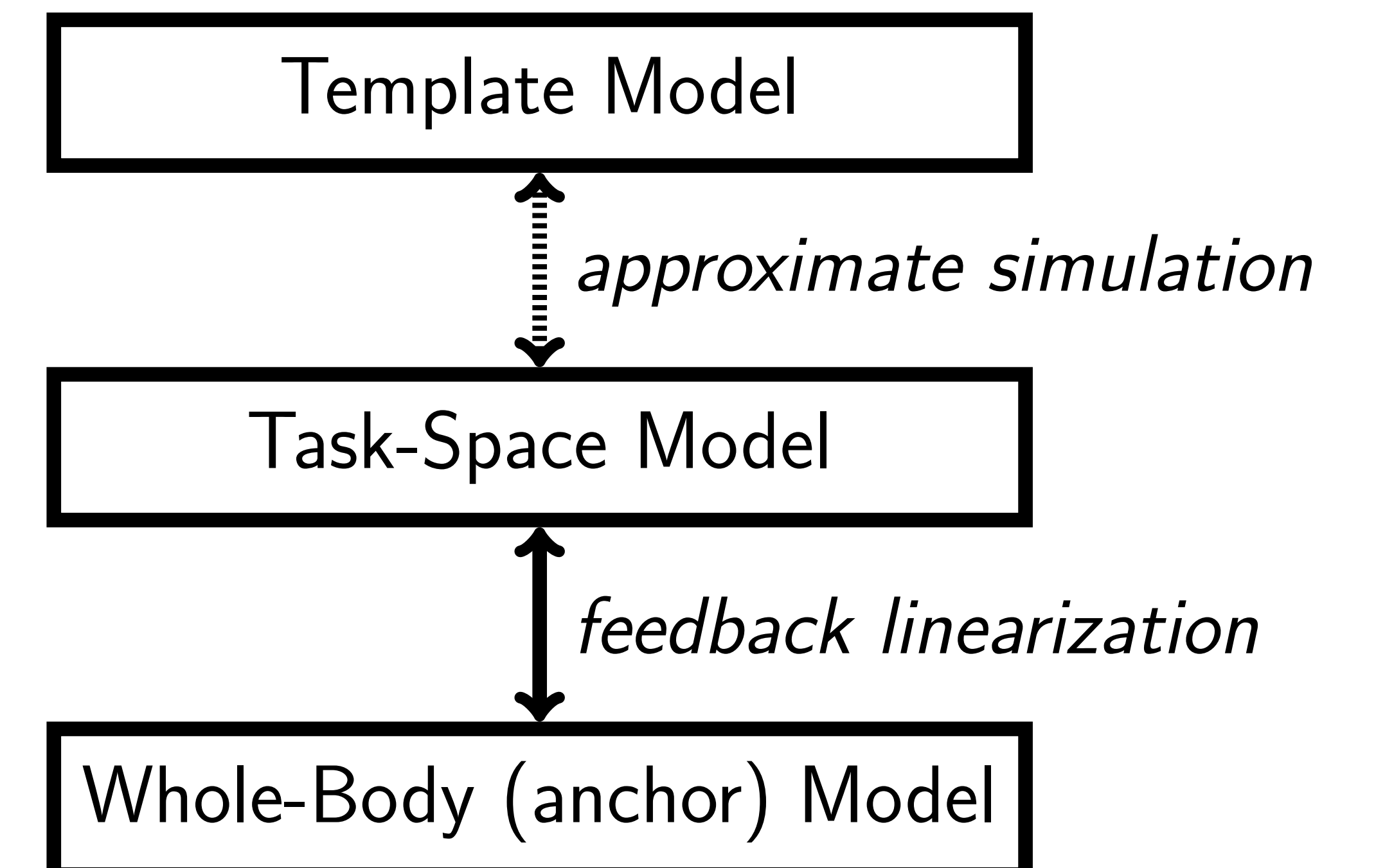
$$u_{\mathcal{V}}(\mathbf{u}_2, \mathbf{x}_1, \mathbf{x}_2) = \mathbf{R}\mathbf{u}_2 + \mathbf{Q}\mathbf{x}_2 + \mathbf{K}(\mathbf{x}_1 - \mathbf{P}\mathbf{x}_2)$$

$$\gamma(\mathbf{u}_2) = \frac{\|\sqrt{\mathbf{M}}(\mathbf{B}_1\mathbf{R} - \mathbf{P}\mathbf{B}_2)\|}{\lambda} \|\mathbf{u}_2\|$$

where

- $\mathbf{P}\mathbf{A}_2 = \mathbf{A}_1\mathbf{P} + \mathbf{B}_1\mathbf{Q}$
- $\mathbf{C}_2 = \mathbf{C}_1\mathbf{P}$
- \mathbf{K} is stabilizing feedback gain for Σ_1
- \mathbf{M} certifies convergence of Σ_1 to zero with rate λ under $\mathbf{u}_1 = \mathbf{K}\mathbf{x}_1$

Templates and Anchors



Template: Linear Inverted Pendulum.

$$\dot{\mathbf{x}}_{lip} = \mathbf{A}_{lip}\mathbf{x}_{lip} + \mathbf{B}_{lip}u_{lip}$$

where u_{lip} is the center-of-pressure position.

Task-Space: Centroidal Dynamics.

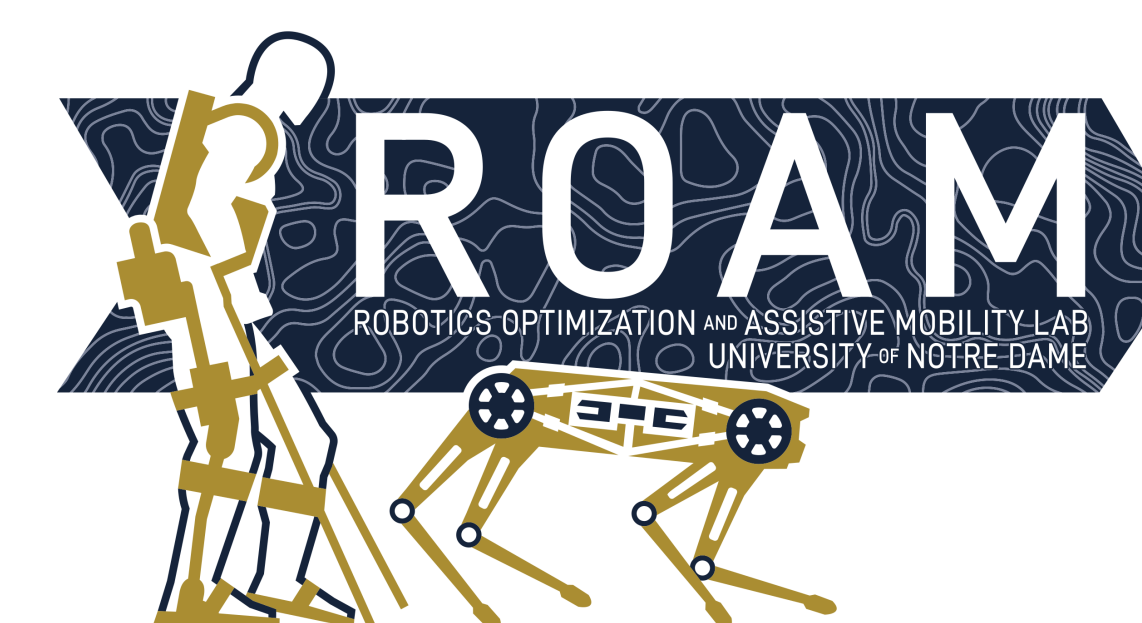
$$\dot{\mathbf{x}}_{task} = \mathbf{A}_{lip}\mathbf{x}_{task} + \mathbf{B}_{lip}u_{task}$$

where

$$\mathbf{x}_{task} = \begin{bmatrix} \mathbf{p}_G \\ \mathbf{h}_G \end{bmatrix} \quad \mathbf{u}_{task} = \dot{\mathbf{h}}_G$$

Anchor: Rigid-Body Model.

$$\mathbf{H}(\mathbf{q})\ddot{\mathbf{q}} + \mathbf{C}(\mathbf{q}, \dot{\mathbf{q}})\dot{\mathbf{q}} + \boldsymbol{\tau}_g = \boldsymbol{\tau}$$

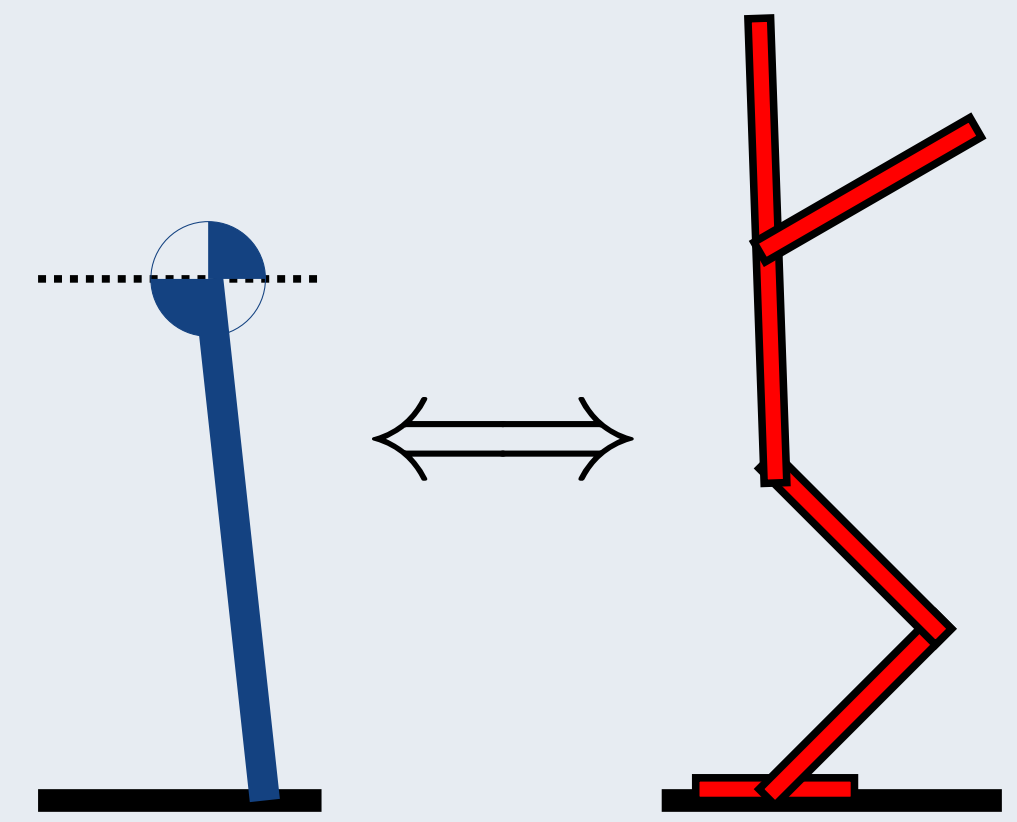


Formal Connections Between Template and Anchor Models

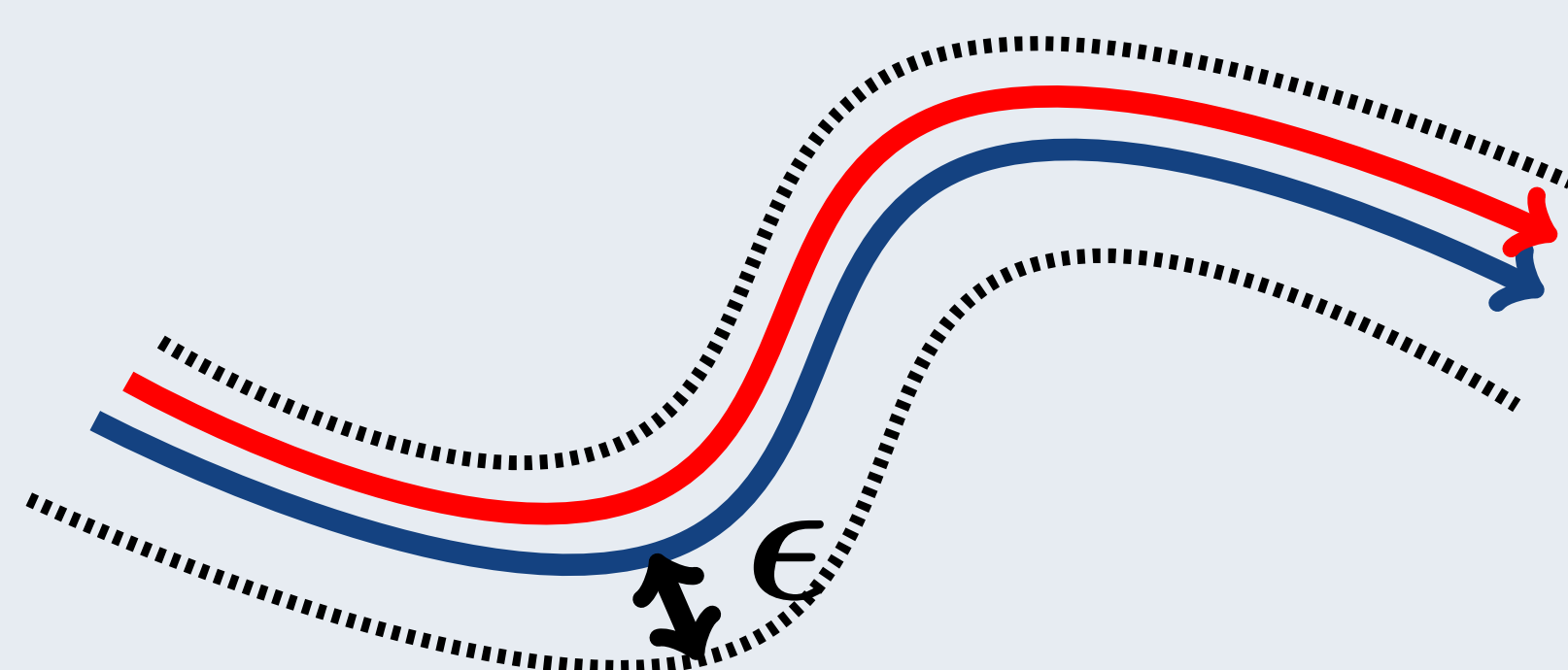
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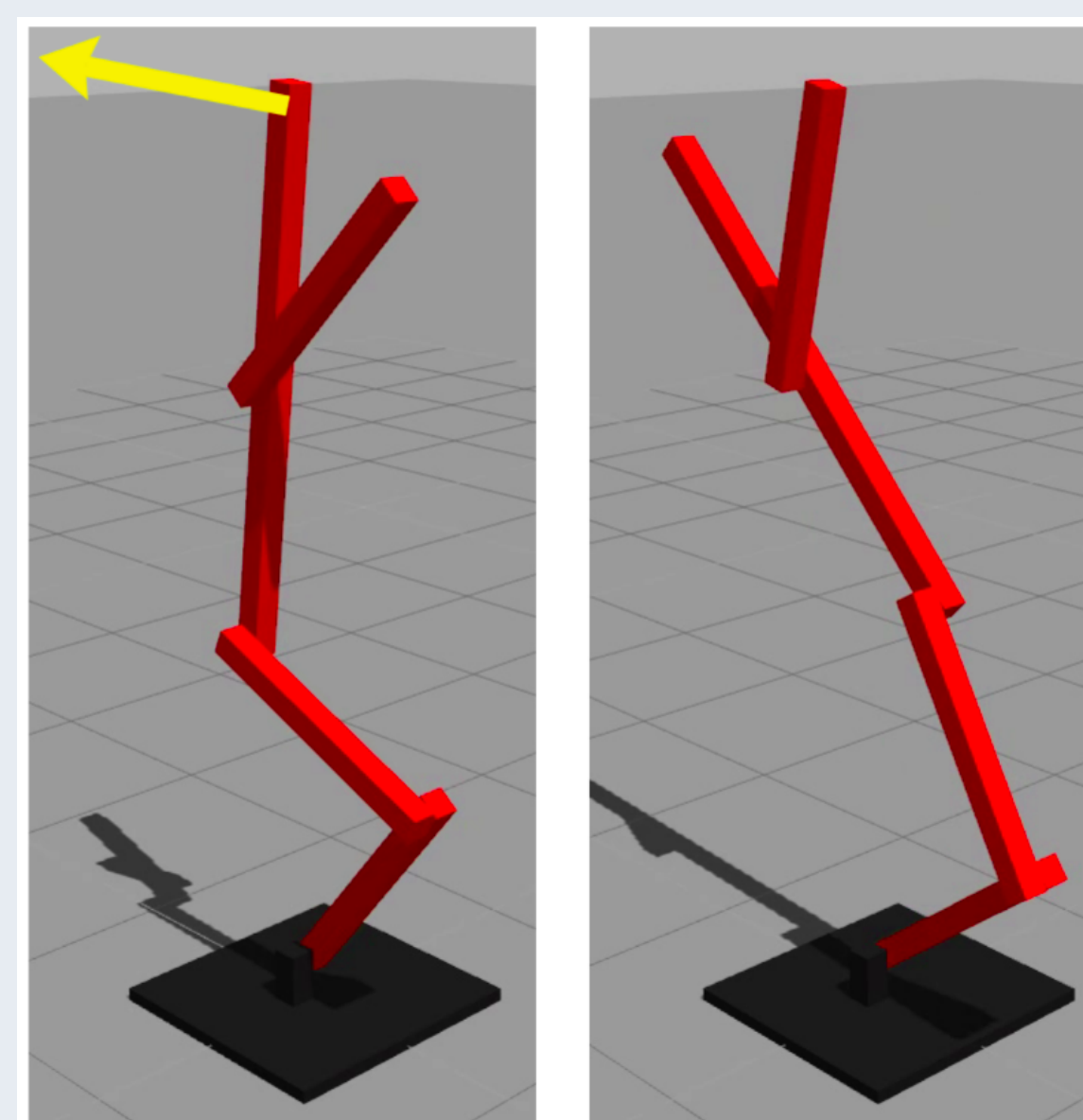
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We establish an **Approximate Simulation Relation** between the **Linear Inverted Pendulum** and a **Planar Balancer**.

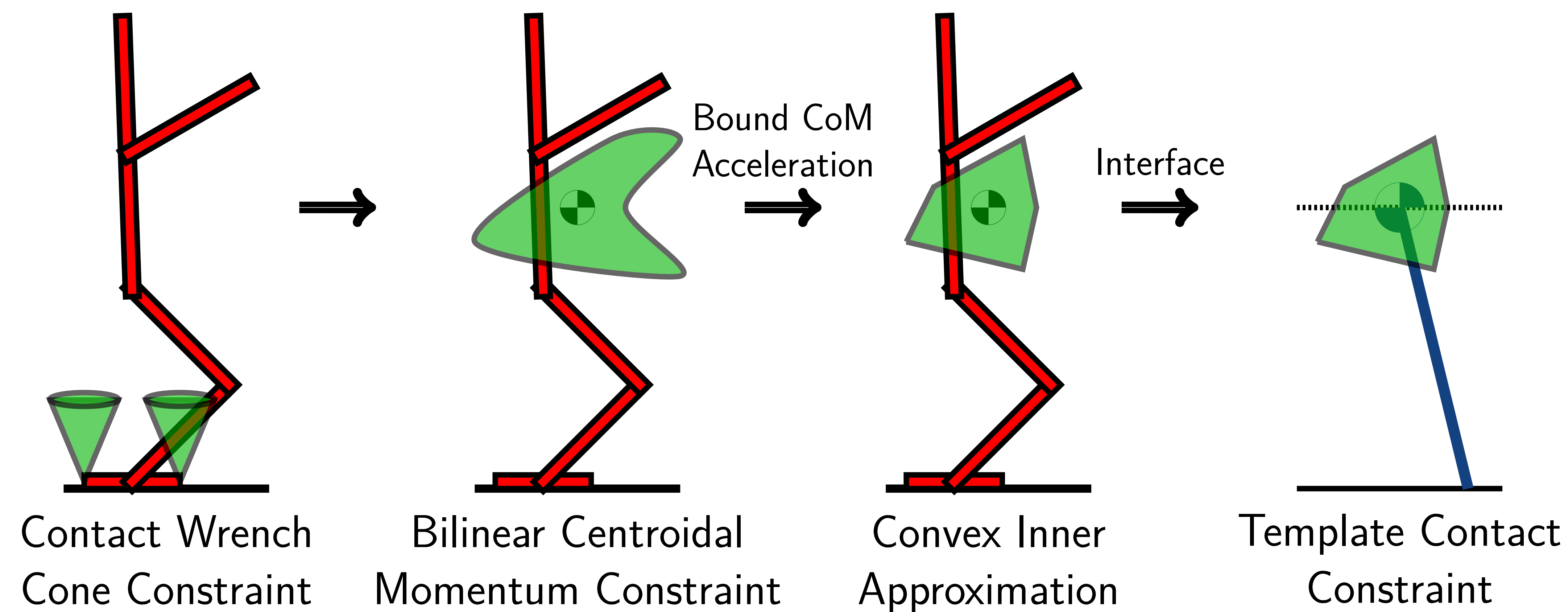


An A.S. relation ensures that the anchor can **track the template with ϵ precision**.



This enables **better push recovery** by allowing centroidal momentum to vary.

Projecting Contact Constraints to the Template



Constraint Linearization

CWC Constraint on contact force:

$$\mathbf{A}\mathbf{f}_0 \leq 0$$

Bilinear constraint on centroidal momentum \mathbf{h}_G :

$$\mathbf{A}^0 \mathbf{X}_G^* (\dot{\mathbf{h}}_G - \begin{bmatrix} \mathbf{0} \\ mg \end{bmatrix}) \leq 0$$

Rewrite in terms of linear (\mathbf{l}_G) and angular (\mathbf{k}_G) momentum:

$$\mathbf{A}^0 \mathbf{X}_G^* \dot{\mathbf{h}}_G \leq \mathbf{A}^0 \mathbf{X}_G^* \begin{bmatrix} \mathbf{0} \\ mg \end{bmatrix}$$

$$\mathbf{A} \begin{bmatrix} \mathbf{I} \\ \mathbf{0} \end{bmatrix} \mathbf{k}_G + \mathbf{A} \begin{bmatrix} \mathbf{S}(\mathbf{p}_G) \\ \mathbf{I} \end{bmatrix} \dot{\mathbf{l}}_G \leq \mathbf{A} \begin{bmatrix} \mathbf{I} & \mathbf{S}(\mathbf{p}_G) \\ \mathbf{0} & \mathbf{I} \end{bmatrix} \begin{bmatrix} \mathbf{0} \\ mg \end{bmatrix}$$

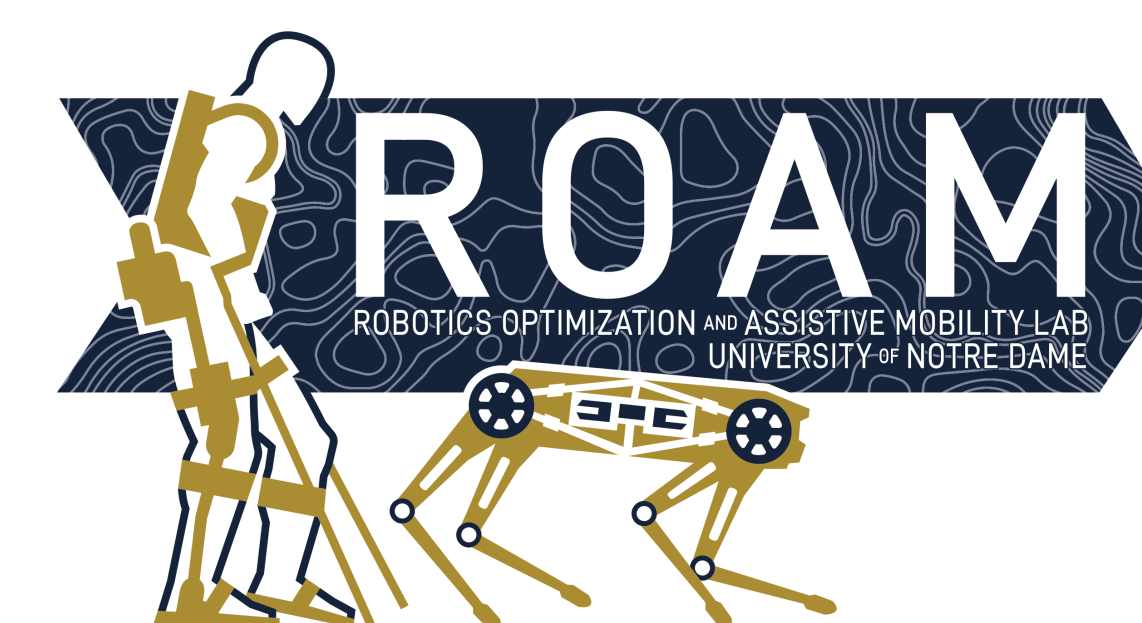
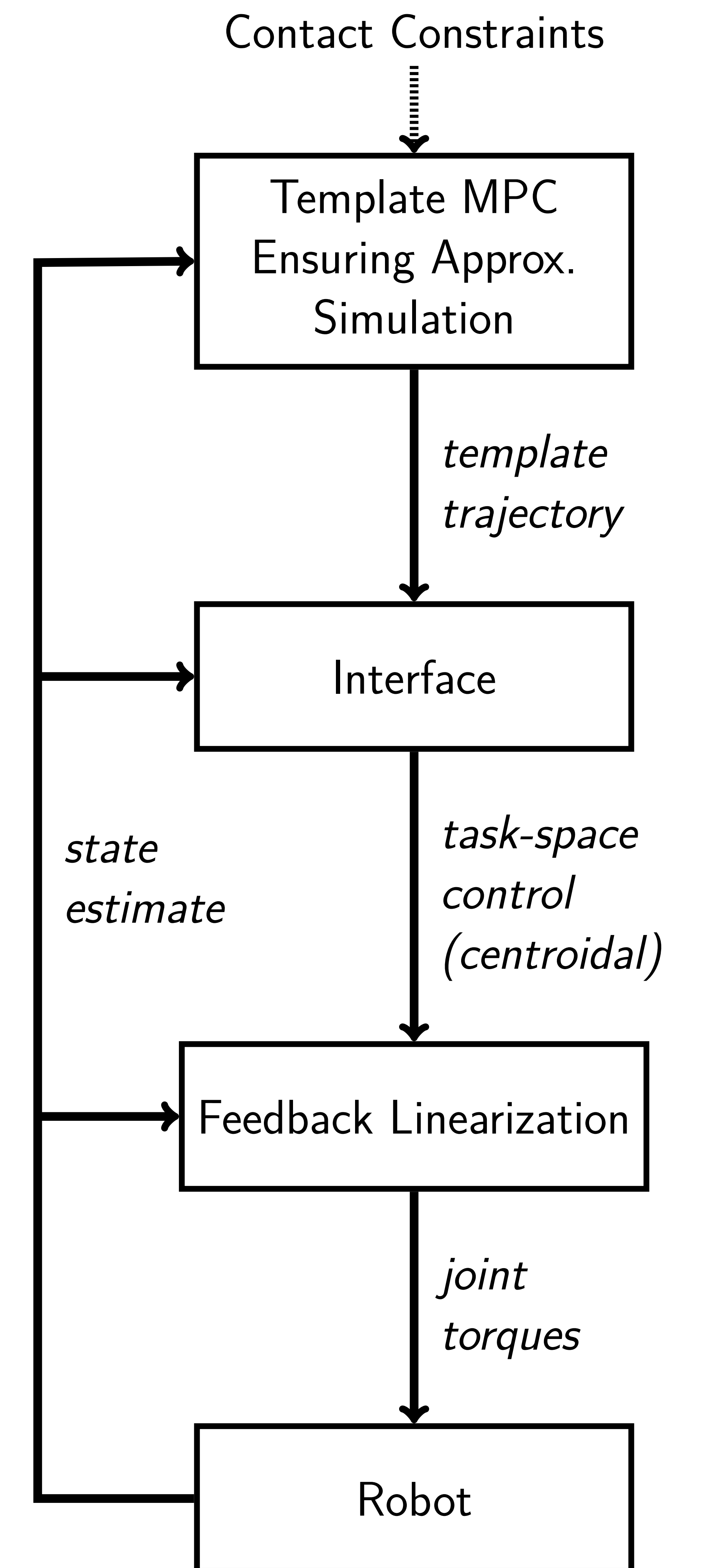
Linear constraint if $\|\dot{\mathbf{l}}_G\|_\infty \leq l_{max}$:

$$\mathbf{A}\mathbf{h}_G + \mathbf{A} \begin{bmatrix} \mathbf{S}(mg) - \mathbf{S}(\dot{\mathbf{l}}_G) \\ \mathbf{0} \end{bmatrix} \mathbf{p}_G \leq \mathbf{A} \begin{bmatrix} \mathbf{0} \\ mg \end{bmatrix}$$

Details

- Template MPC accounts for friction
- Interface enforces formal relation with centroidal dynamics
- Feedback linearization determines joint torques, no need to consider friction

Our Control Framework

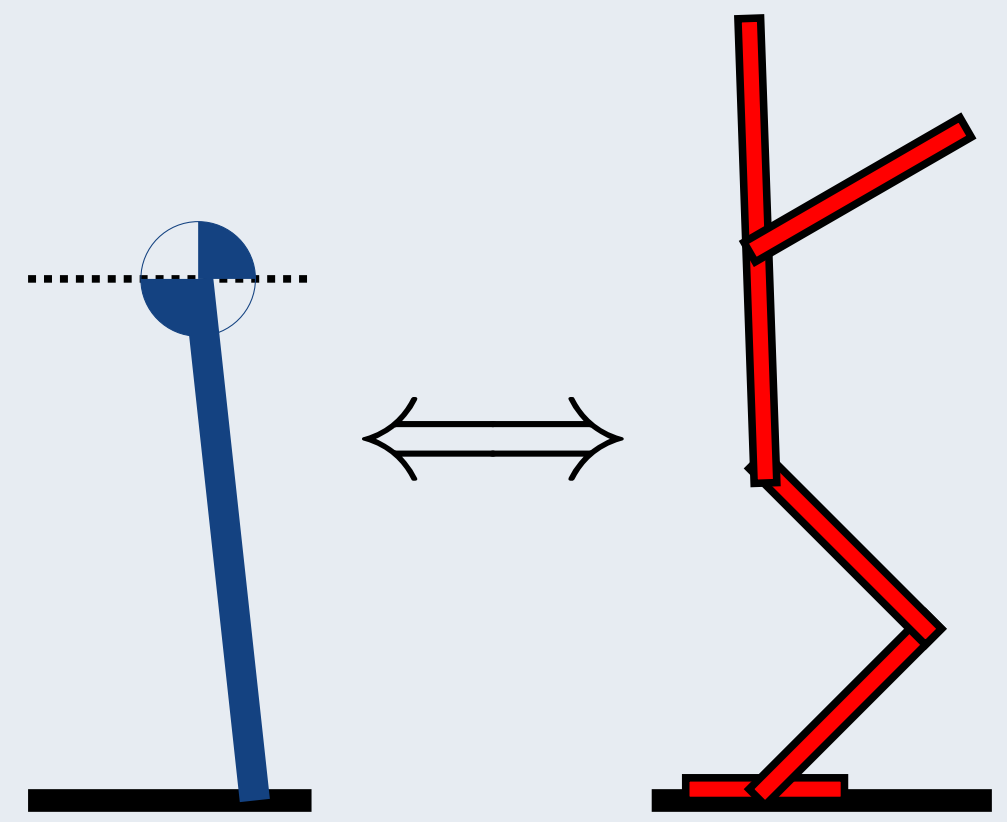


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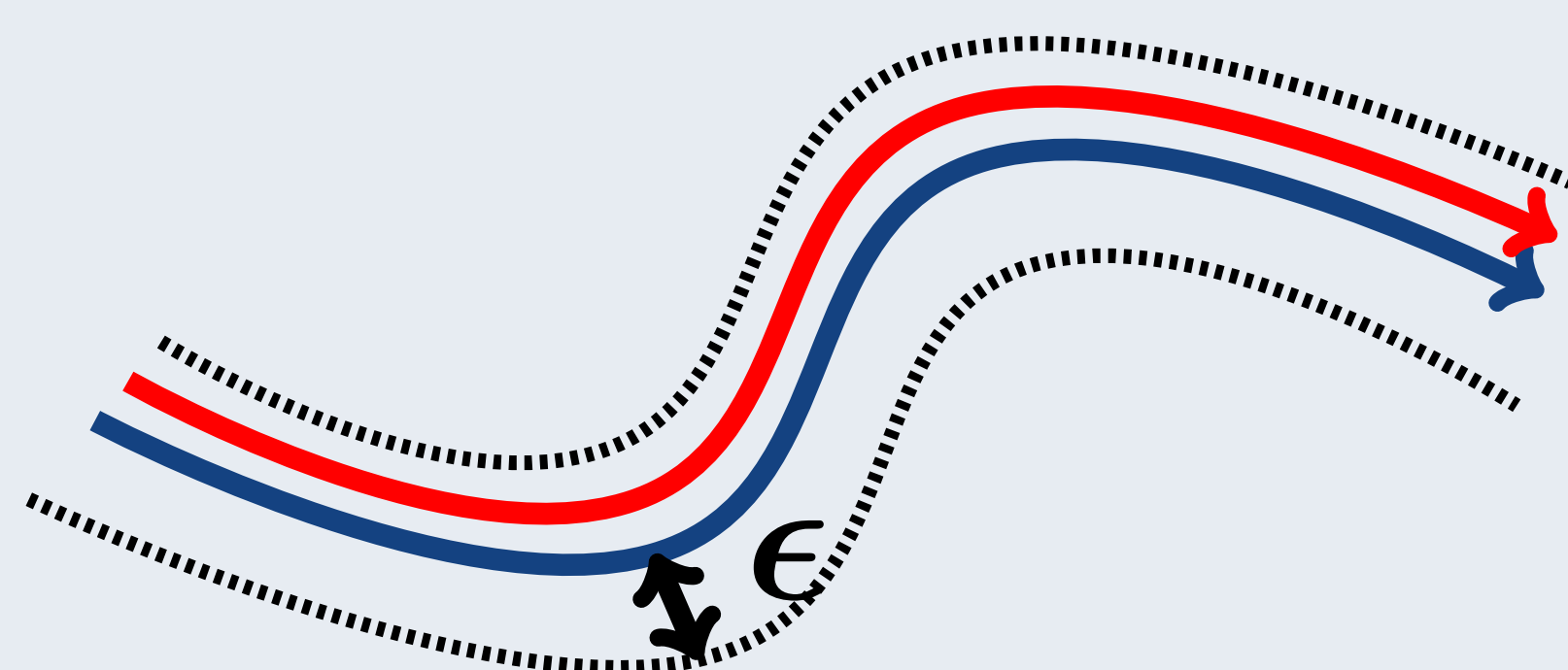
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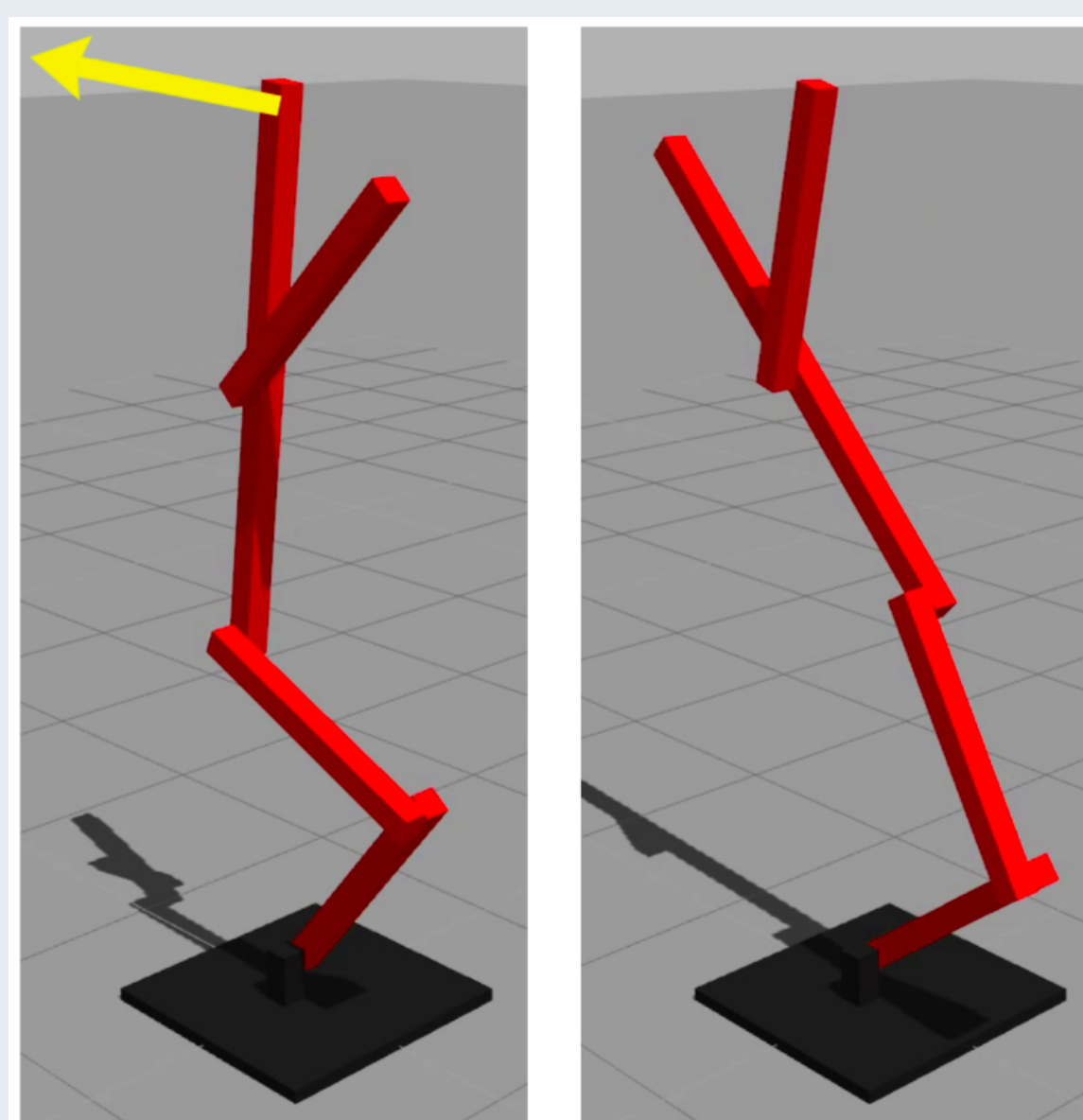
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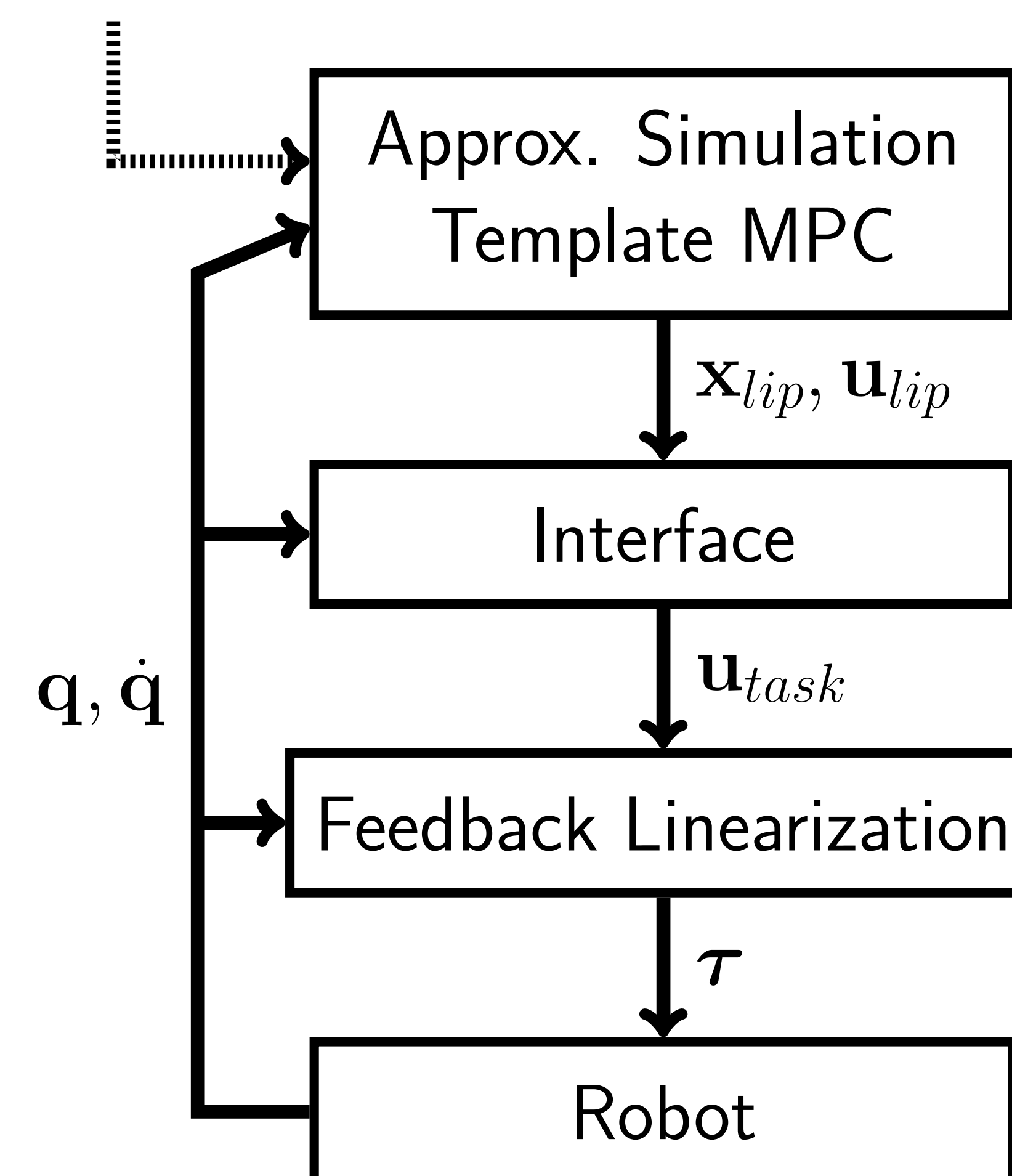
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Proposed Control Framework

Contact Constraints



Contact-aware Template MPC

$$\min \sum_{t=0}^{N-1} L(\mathbf{x}_{lip}(t), \mathbf{u}_{lip}(t)) + L_f(\mathbf{x}_{lip}(N))$$

s.t. $\mathbf{x}_{lip}(0), \mathbf{x}_{task}(0)$ given

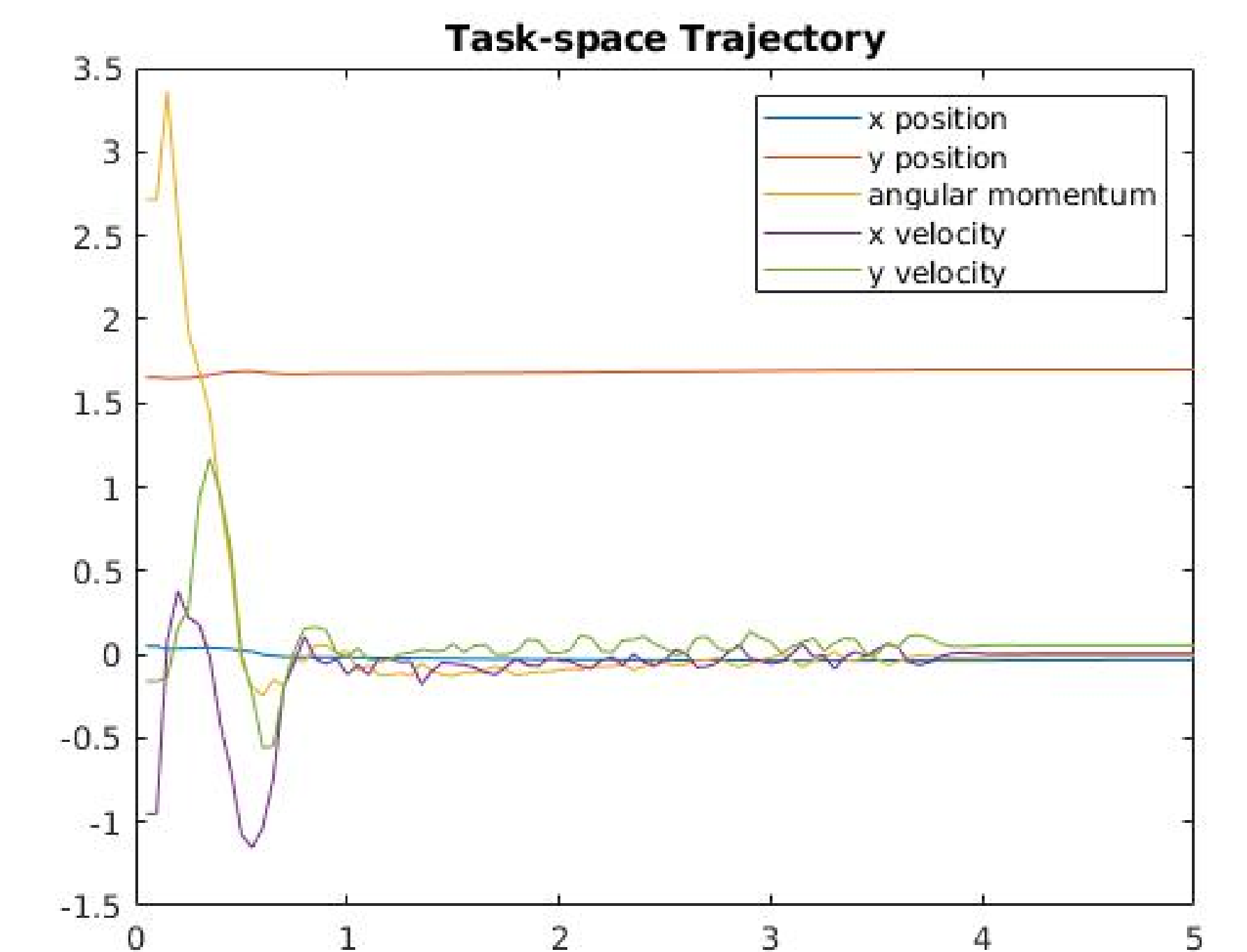
$$\dot{\mathbf{x}}_{lip} = \mathbf{A}_{lip}\mathbf{x}_{lip} + \mathbf{B}_{lip}\mathbf{u}_{lip}$$

$$\dot{\mathbf{x}}_{task} = \mathbf{A}_{task}\mathbf{x}_{task} + \mathbf{B}_{task}\mathbf{u}_{task}$$

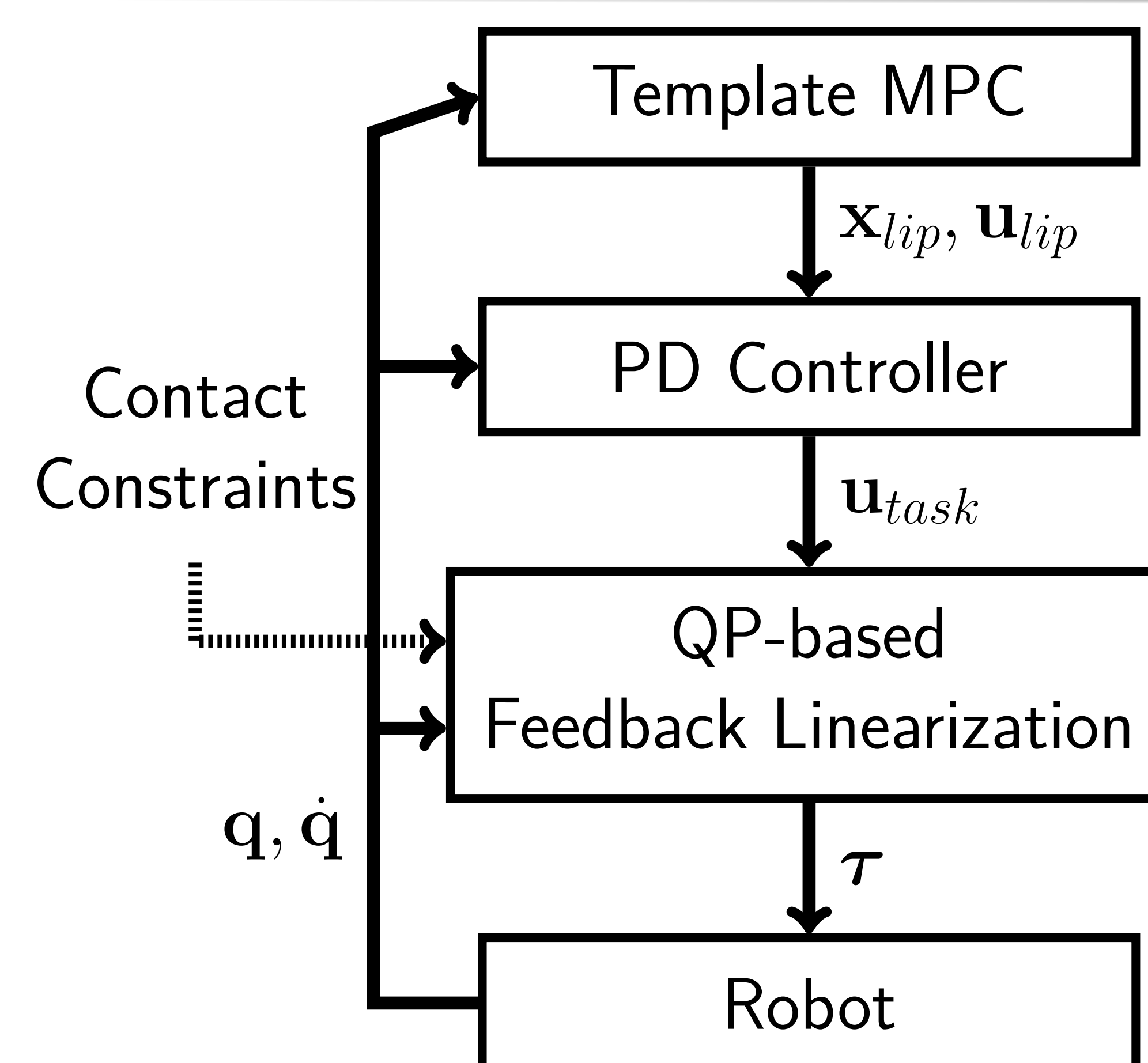
$$\mathbf{u}_{task} = \mathbf{R}\mathbf{u}_{lip} + \mathbf{Q}\mathbf{x}_{lip} + \mathbf{K}(\mathbf{x}_{task} - \mathbf{P}\mathbf{x}_{lip})$$

$$\mathbf{A}_{cwc} \begin{bmatrix} \mathbf{x}_{task} \\ \mathbf{u}_{task} \end{bmatrix} \leq \mathbf{b}_{cwc}$$

$$\|\dot{\mathbf{i}}_G\|_{\infty} \leq \dot{i}_{max}$$



Standard Control Framework

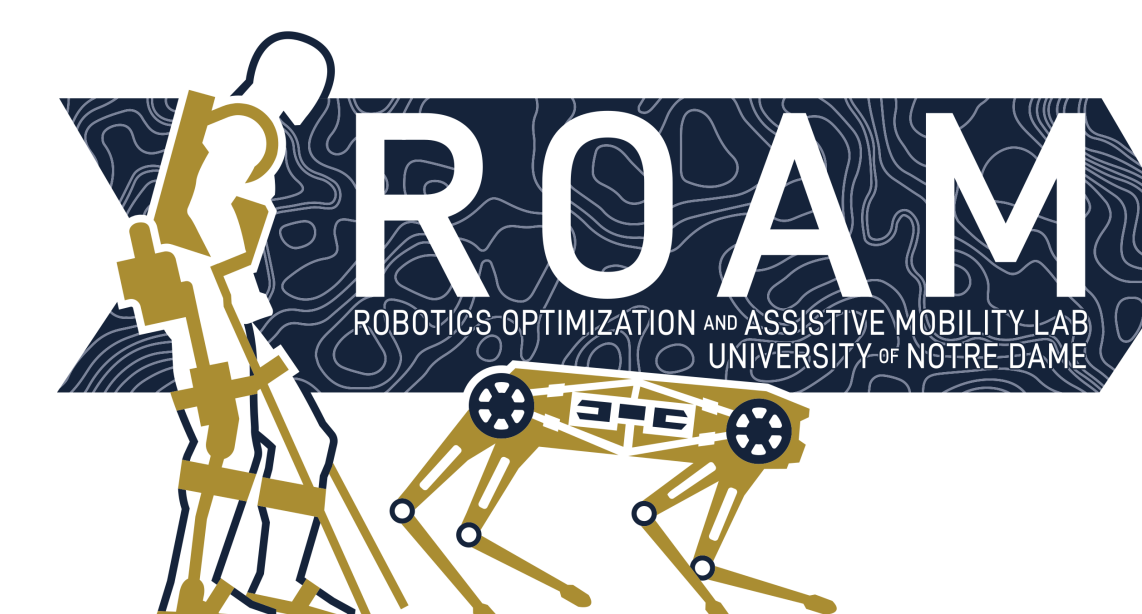
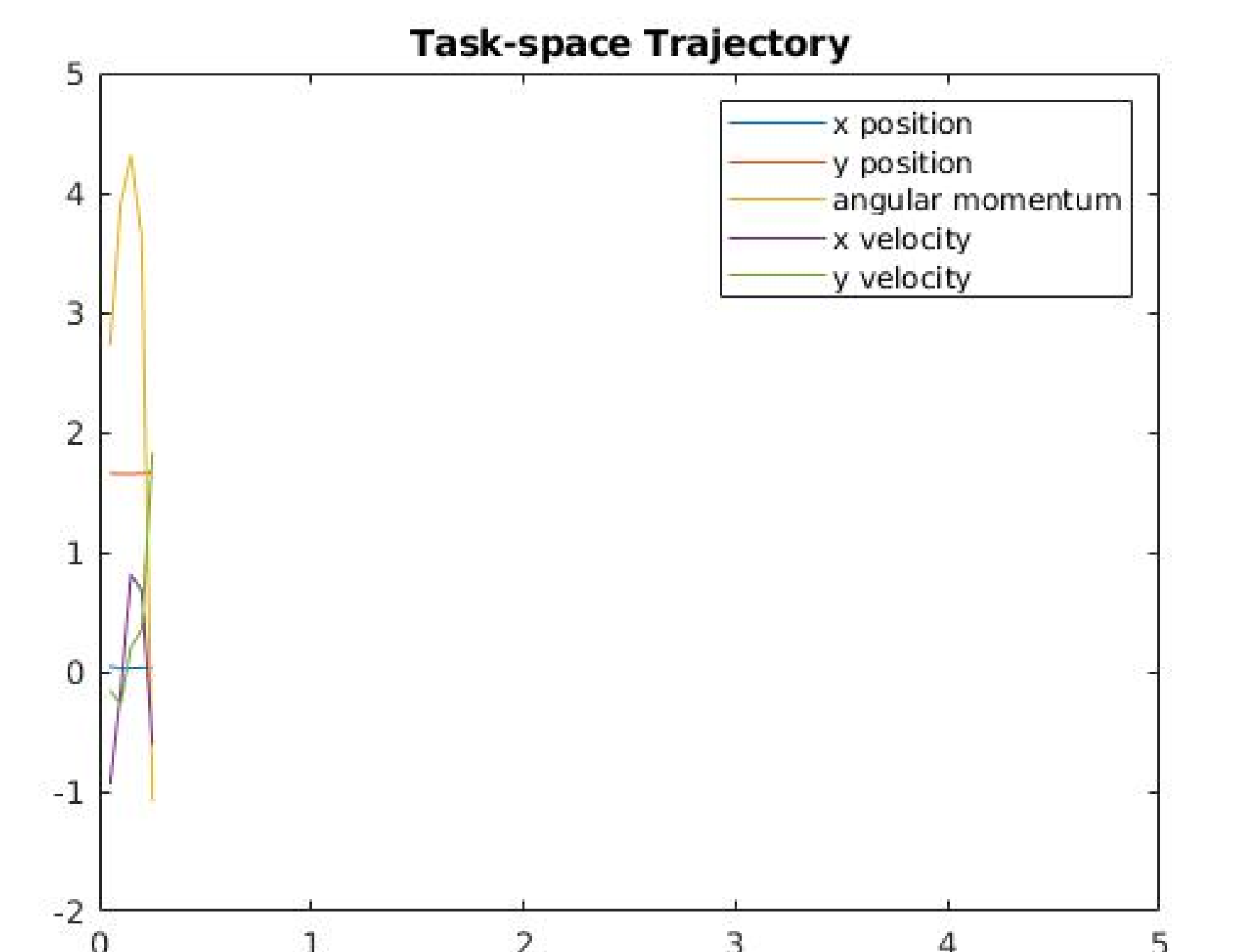


Traditional Template MPC

$$\min \sum_{t=0}^{N-1} L(\mathbf{x}_{lip}(t), \mathbf{u}_{lip}(t)) + L_f(\mathbf{x}_{lip}(N))$$

s.t. $\mathbf{x}_{lip}(0), \mathbf{x}_{task}(0)$ given

$$\dot{\mathbf{x}}_{lip} = \mathbf{A}_{lip}\mathbf{x}_{lip} + \mathbf{B}_{lip}\mathbf{u}_{lip}$$

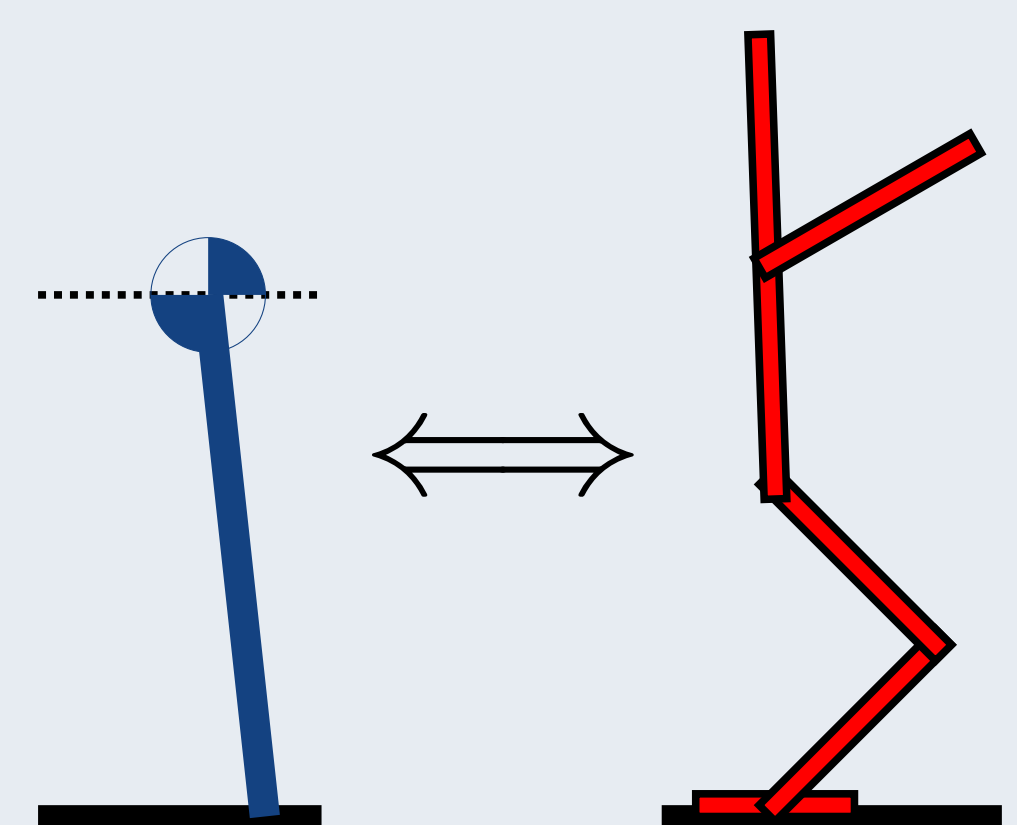


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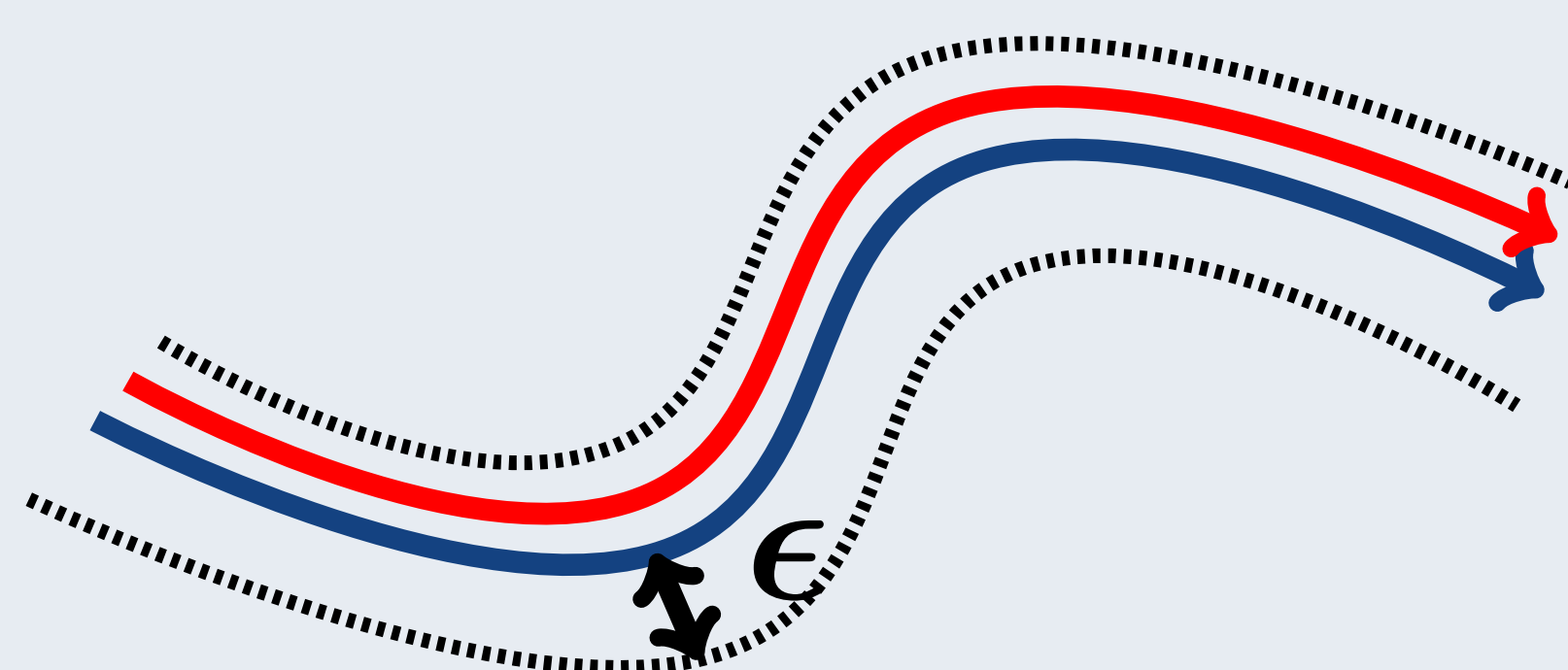
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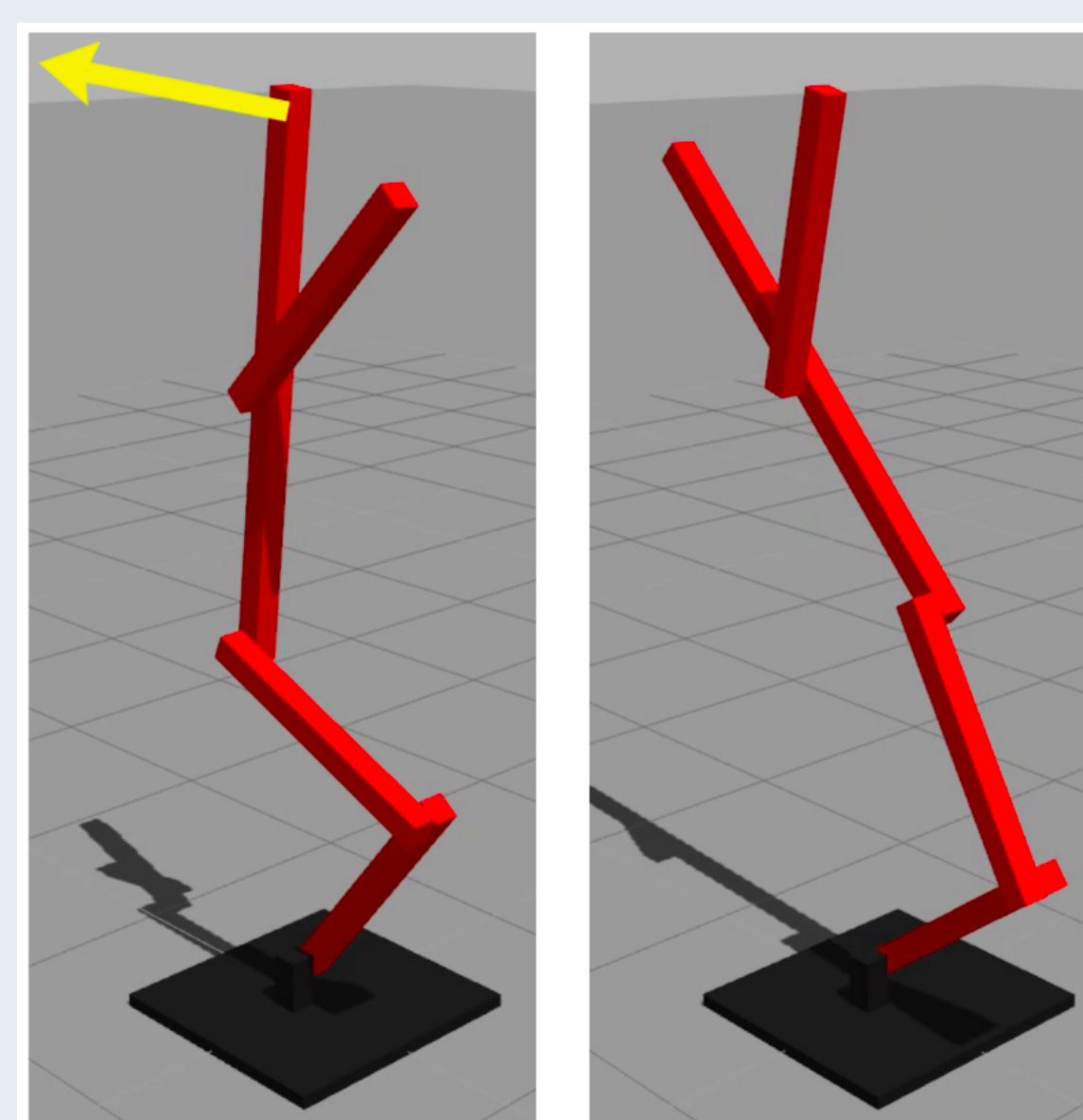
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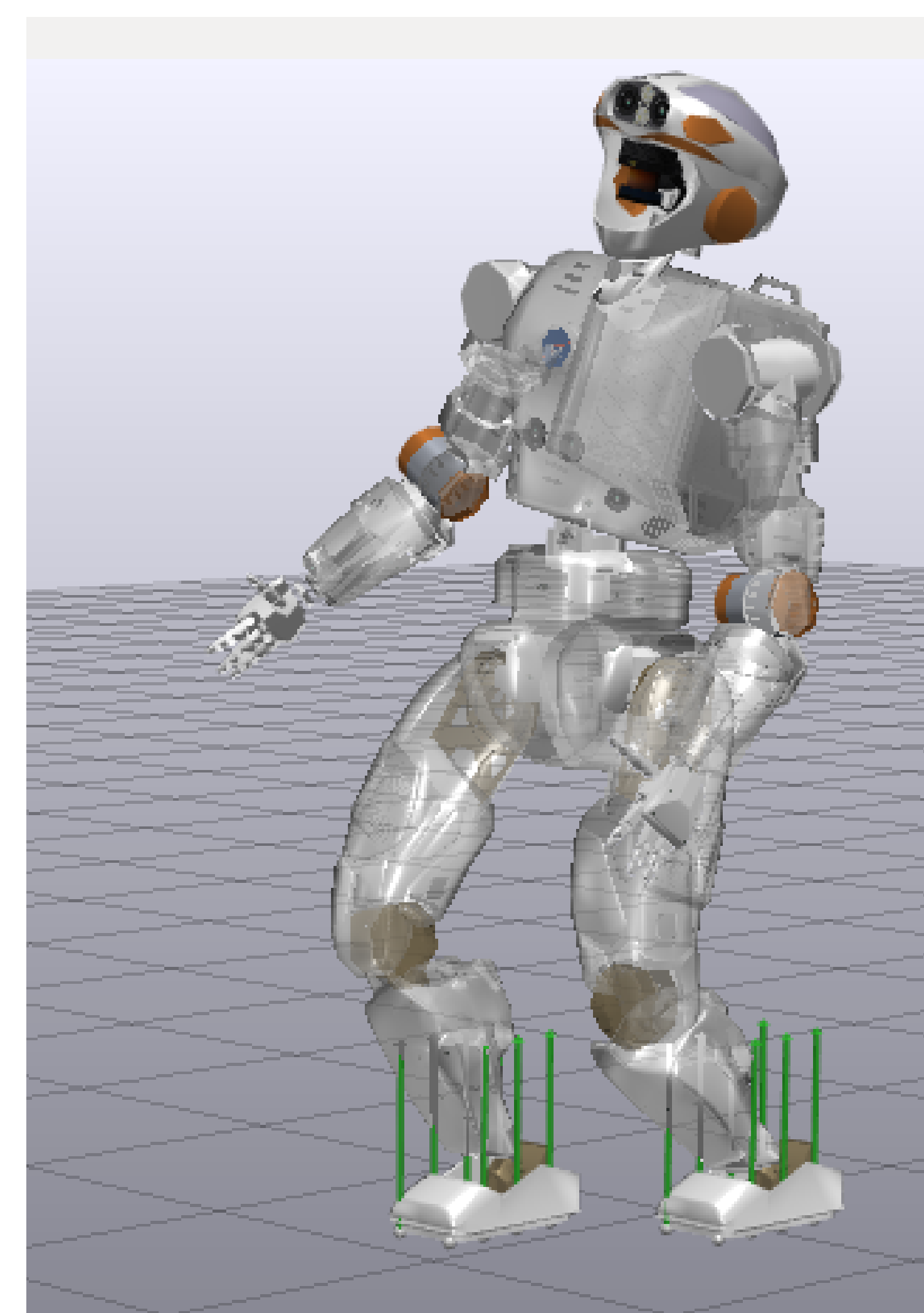
Future Work: Walking Control

Challenges:

- Impact disturbances: robust approximate simulation
- Multiple support

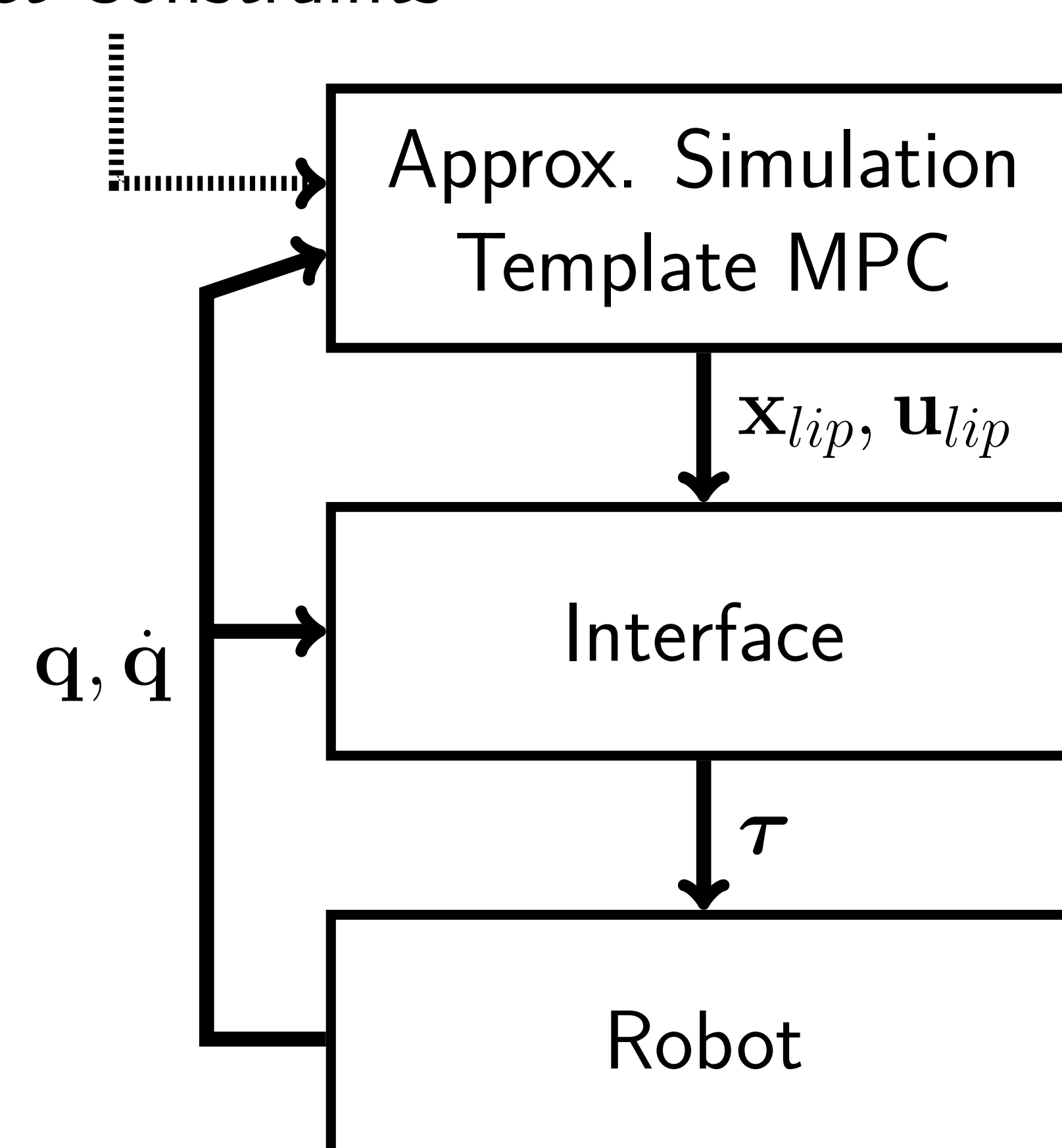
Anticipated Benefits:

- Template ensures contact feasibility
- Less parameter tuning
- More robust



Future Work: Beyond Task-Space Approximate Simulation

Contact Constraints



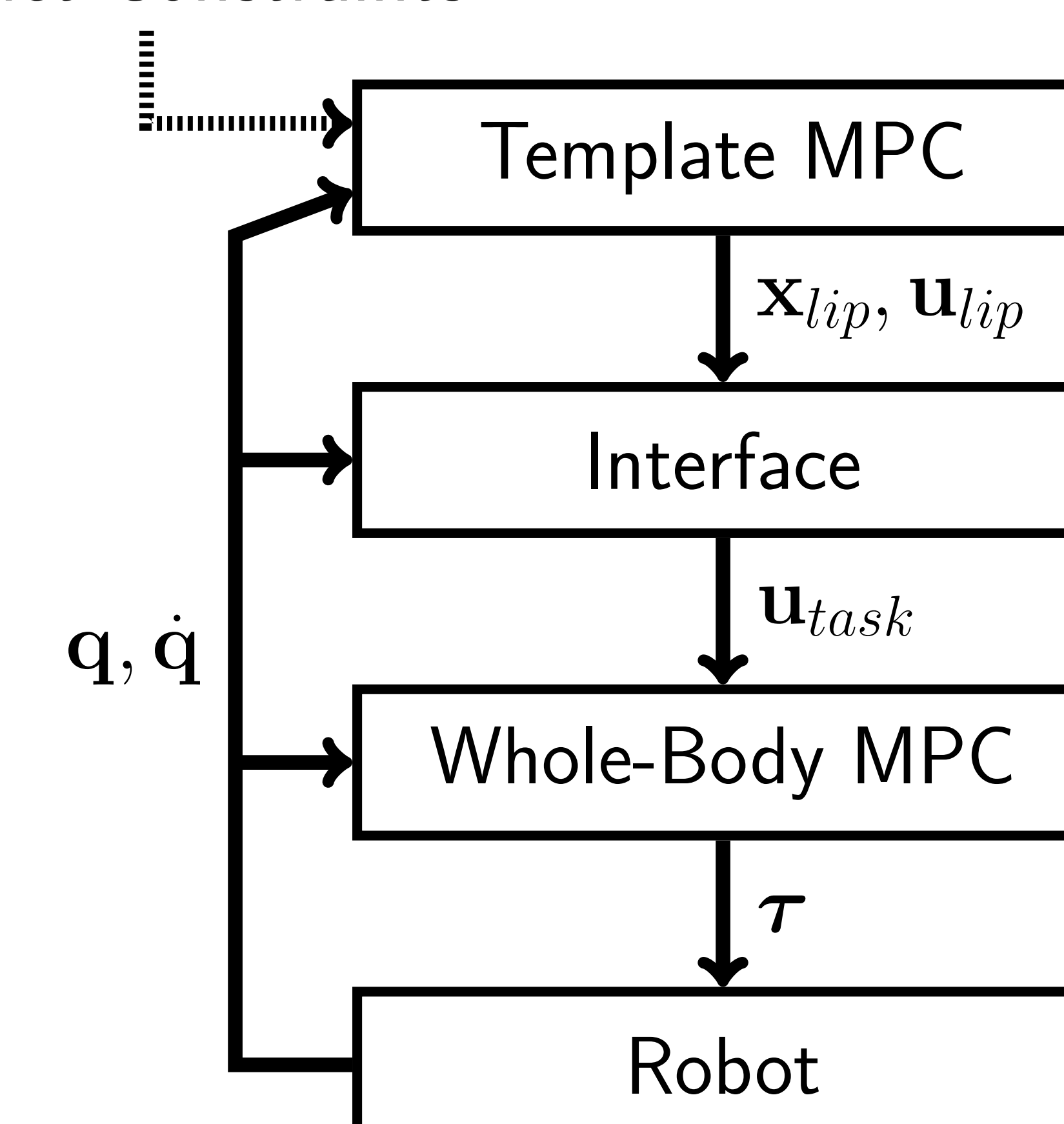
Goal: establish formal connection with whole-body dynamics directly

- Use SOS techniques to find $\mathcal{V}(\mathbf{x}_1, \mathbf{x}_2)$
- D-SOS/SD-SOS to address scalability challenges [1]
- Interface = closed-form $(\mathbf{x}_{lip}, \mathbf{u}_{lip}) \mapsto \boldsymbol{\tau}$

Future Work: Enabling Whole-Body MPC

DDP is promising, but friction constraints are difficult to handle [3].

Contact Constraints



Use approximate simulation to enforce contact constraints with the template.

References

- [1] A. A. Ahmadi and A. Majumdar. *SIAM Journal on Applied Algebra and Geometry*, 3(2):193–230, 2019.
- [2] A. Girard and G. J. Pappas. *Automatica*, 45(2):566–571, 2009.
- [3] Y. Tassa, T. Erez, and E. Todorov. In *2012 IEEE/RSJ International Conference on Intelligent Robots and Systems*, pages 4906–4913. IEEE, 2012.

